Cyber-Physical Systems

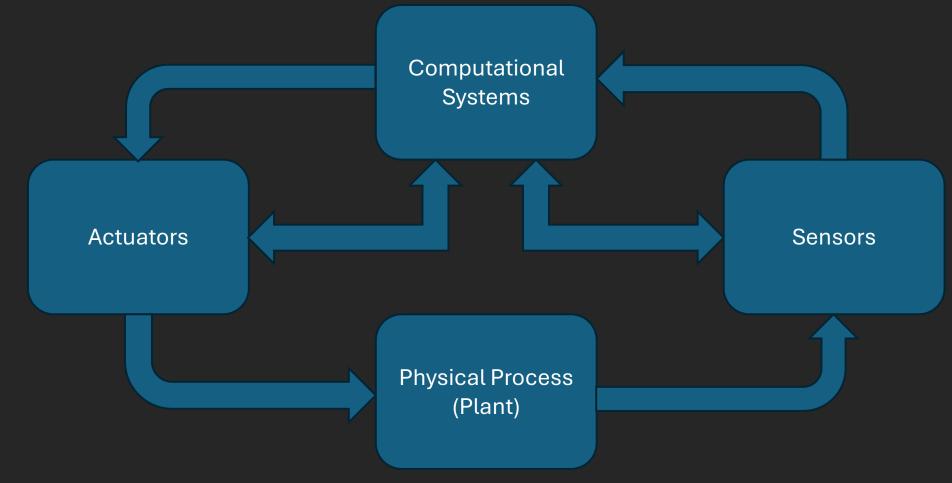
Dr. Jonathan Jaramillo



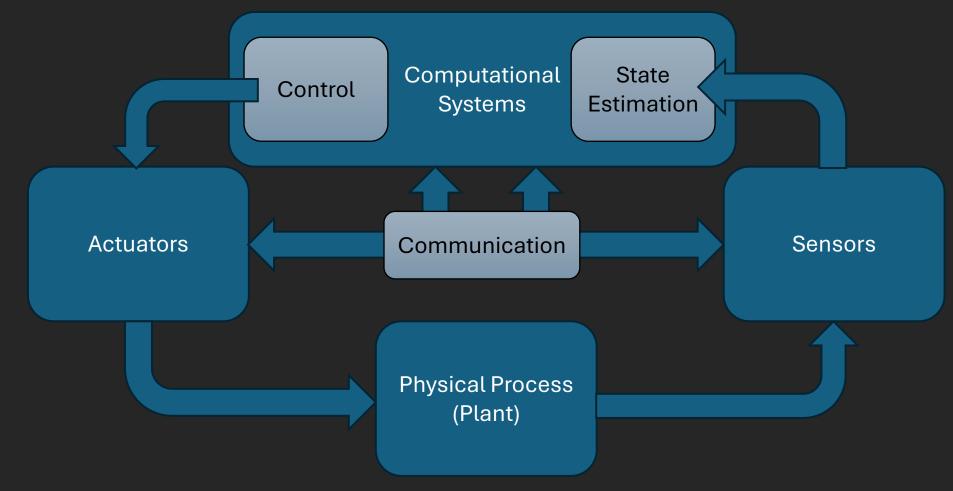
Feedbacl Control



What are Cyber-Physical Systems?



What are Cyber-Physical Systems?



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Feedback Control

- Feedback control is a method of automatically regulating a system by continuously measuring its output, comparing it to a desired reference, and adjusting inputs to minimize the error
- Feedback control in Cyber-physical Systems
 - Computational components
 - System Models
 - State Estimation

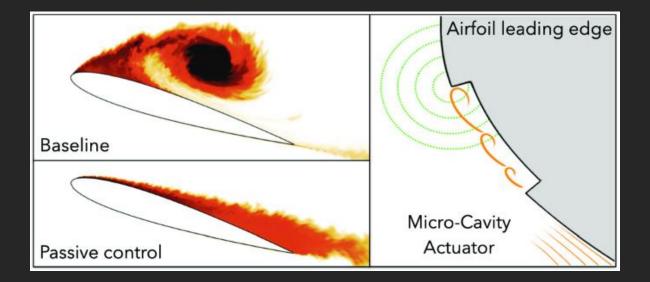
Non-Feedback Control

• Timers

- Rotational Stability Spinning frisbee
- Passive drag elements (arrows and planes)
- Boiling food







Feedback Control

- Thermostat
- Cruise control
- Autofocus

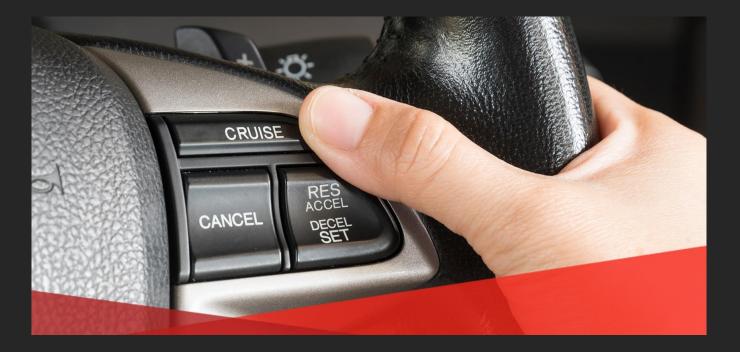
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• Computer fan speed





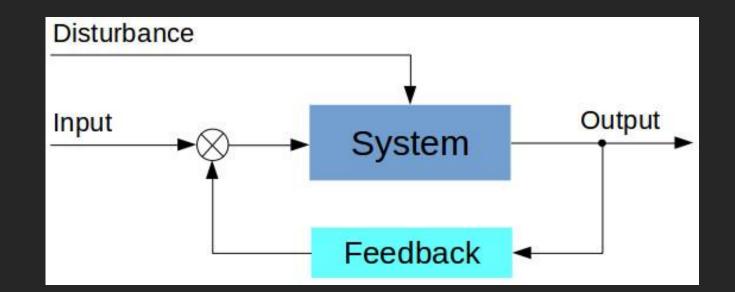




Concepts of Feedback Control

- State-space control
- PID Control

- Other Control Concepts
 - Adaptive Control
 - Model Predictive Control
 - Distributed Control





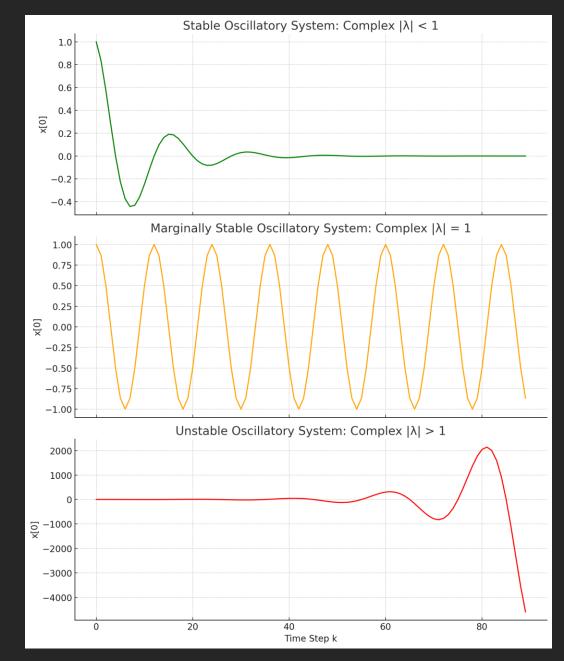
- Canonical State Space Form (Assume full Observability)
 - $x_{k+1} = Ax_k + Bu_k$
 - $y_{k+1} = x_{k+1}$
- With no input system dynamics are governed by A for all time
 - $x_{k+1} = Ax_k$
- Stability analysis

- Eigan values of A are given by $det(\lambda I A) = 0$
- Unstable if $|\lambda_i| > 1$ for all λ_i
- Marginally Stable if at least one $|\lambda_i|=1\,$ and all other $|\lambda_i|\leq 1\,$
- Stable if all $|\lambda_i| < 1$

- $x_{k+1} = Ax_k$
- Stability analysis
 - $|\lambda_i| > 1$ for all λ_i
 - At least one $|\lambda_i| = 1$, all other $|\lambda_i| \leq 1$
 - $|\lambda_i| < 1$

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 If stable, system will converge to dominate eigen value



Can We Change the Eigan Values?

- The system dynamics can't change without altering the system.
- Create a control law
 - $x_{k+1} = Ax_k + Bu_k$
 - $u_k = -Kx_k$

- $x_{k+1} = Ax_k BKx_k = (A BK)x_k$
- Now system dynamics are governed by A BK and its eigen values

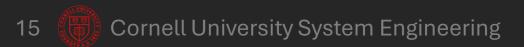
Control in State Space Methods

• Pole Placement

- Choosing K such that the system has the exact dynamics you want
- Linear Quadratic Regulator
 - Choses *K* to minimize a loss function
 - $J = \int_0^\infty x(t)^T Q x(t) + u(t)^T R u(t) dt$
 - Q penalizes deviation in the state
 - *R* penalizes control effort

- Advantages
 - Tools for fine tuning exact system performance exist
 - Simple, predicable output
 - Works for multidimensional control
- Disadvantages
 - Requires exact knowledge of system
 - Errors in system dynamics could result in unstable outputs
 - Susceptible to noise
 - Not robust

Model Predative Control



- Linear Quadratic Regulator
 - Choses *K* to minimize a loss function
 - $J = \int_0^\infty x(t)^T Q x(t) + u(t)^T R u(t) dt$
 - Q penalizes deviation in the state
 - *R* penalizes control effort
- Time horizon is infinite
- Once K is calculated, it is applied for all time

Discrete Finite Time Horizon

- x[k+1] = f(x[k], u[k])
- Look at a finite amount of time into the future

•
$$u^* = \arg \min_{u[0...N-1]} \sum_{i=0}^{N-1} (x[i] - x_{ref}[i])^T Q(x[i] - x_{ref}[i]) + u[i]^T Ru[i])$$

•
$$u^* = \arg \min_{u[0...N-1]} \sum_{i=0}^{N-1} x[i]_{error}^T Q x_{error} + u[i]^T R u[i])$$

Choosing Q & R

•
$$Q = \begin{bmatrix} q_1 & 0 & \cdots & 0 \\ 0 & q_2 & \cdots & 0 \\ \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & \cdots & q_n \end{bmatrix}$$

• $R = \begin{bmatrix} r_1 & 0 & \cdots & 0 \\ 0 & r_2 & \cdots & 0 \\ \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & \cdots & r_n \end{bmatrix}$

Choosing Q & R

- Usually, Q & R are diagonal because state variables are independent
- When to use non-diagonal matrices
 - Coupled state variables
 - Coupled control variables
- Example of coupled state variables
 - Angular speed and linear speed

•
$$x = \begin{bmatrix} x \\ y \\ \theta \\ v \\ \dot{\theta} \end{bmatrix}, Q = \begin{bmatrix} q_x & 0 & 0 & 0 & 0 \\ 0 & q_y & 0 & 0 & 0 \\ 0 & 0 & q_\theta & q_{v\theta} & 0 \\ 0 & 0 & q_{v\theta} & q_v & q_{v\dot{\theta}} \\ 0 & 0 & 0 & q_{v\dot{\theta}} & q_{\dot{v}} & q_{\dot{\theta}} \end{bmatrix}$$

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Model Predictive Control

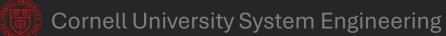
Advantages

- Handles multi-input, multi-output systems
- Handles non-linear systems
- Receding horizon approach and accurate system prediction
- Disadvantages
 - Very computationally expensive
 - Errors in system dynamics could result in unstable outputs
 - Tuning complexity
 - Implementation complexity



Model Predictive Control





PID Control



Proportional Control

- Calculate error
 - $e(t) = x_{ref}(t) x(t)$
- Calculate control value
 - $u(t) = K_P e(t)$
- What happens when there is a force resisting the system?
- Proportional control will always have steady state error



Proportional-Integral Control

- Calculate error
 - $e(t) = x_{ref}(t) x(t)$
- Calculate control value
 - $u(t) = K_P e(t) + K_I \int_0^t e(\tau) d\tau$
- Removes steady state error!
- Introduces overshoot.

Proportional-Integral-Derivative Control

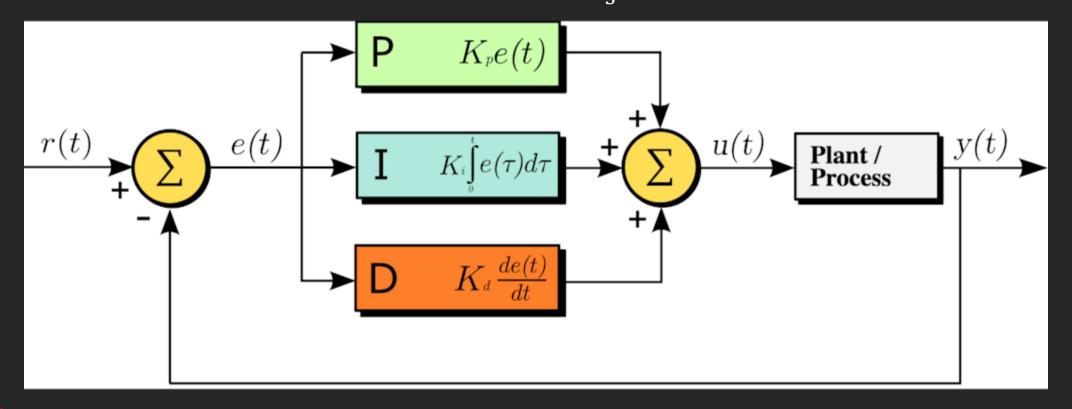
- Calculate error
 - $e(t) = x_{ref}(t) x(t)$
- Calculate control value

•
$$u(t) = K_P e(t) + K_I \int_0^t e(\tau) d\tau + K_D \frac{d}{dt} e(t)$$

- Can reduce overshoot
- Can produce instability

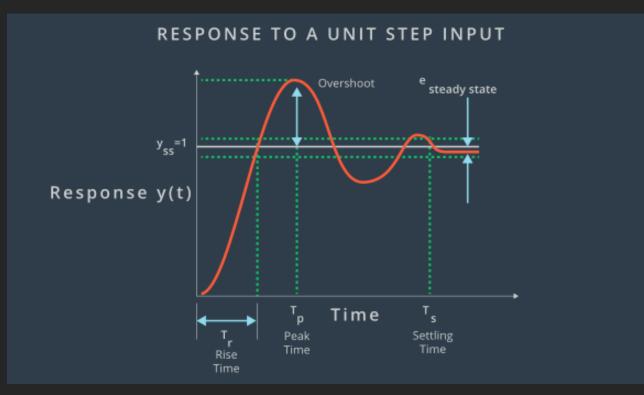
PID Control in Discrete Time

• $u[k] = K_P + K_I \sum_{i=0}^{k} e[i]T_s + K_D \frac{e[k] - e[k-1]}{T_s}$



Characterizing a PID Controller

- Rise Time
- Settling Time
- Overshoot
- Steady State Error





PID Parameters

• K_P

- Faster response, more reactive
- Overshoot and oscillation
- *K*_{*I*}
 - Eliminates steady-state error
 - Sluggish, integrator windup
- *K*_D

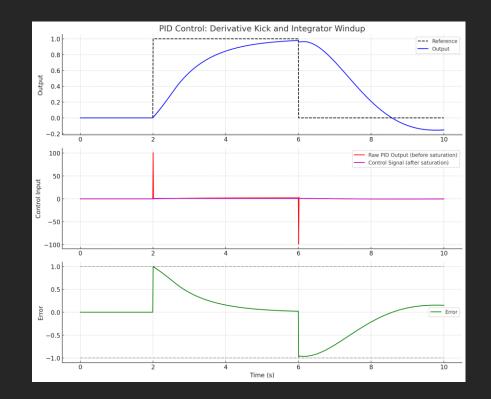
- Reduces overshoot, adds dampening
- Sensitive to noise

Potential Problems with PID

- Constant steady-state error
 - Integrator windup
- Derivative Kick
 - High frequency noise
 - Step change in reference signal
- Output Clamping

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• Calculate u(t) is greater than actuator can generate



Derivative Kick

•
$$\frac{de}{dt} = \frac{d}{dt} \left(x_{ref}(t) - x(t) \right)$$

• $\frac{de}{dt} = \frac{dx_{ref}(t)}{dt} - \frac{dx(t)}{dt}$

• $\frac{dx_{ref}(t)}{dt}$ will spike if there is a change in reference signal

•
$$\frac{de}{dt} = -\frac{dx(t)}{dt}$$

Manually Tuning PID Controllers

- Start with $K_I = 0$ and $K_D = 0$
- Increase K_P until system begins to oscillate
- Add K_D to reduce overshoot and dampen oscillations
- Add *K_I* to eliminate steady-state error
- Repeat

Ziegler-Nichols Method

- Set $K_I = 0$ and $K_D = 0$
- Increase K_P until system starts to oscillate
 - This gain is K_w and period of oscillation is T_w

Controller	K _P	K _I	K _D
Р	$0.5K_w$	-	-
PI	$0.45K_{w}$	$1.2K_P/T_w$	-
PID	0.6 <i>K</i> _w	$2K_P/T_w$	$K_p T_w/8$



PID Control

- Advantages
 - Simple doesn't require detailed system modeling or design
 - Computationally efficient
 - Effective for many single input-single output systems
- Disadvantages

- Limited in multi-variable systems
- Poor predictive capability
- Sensitive to tuning and nonlinearities



- Challenges in implementing feedback control systems
 - Unknown system dynamics
 - Changing system dynamics
- Examples

- Vehicles carry different payloads
- Electrical grid demand changes from season to season
- User of a system might change

- $x_{k+1} = A_k x_k + B_k u_k$
- Adaptive control
 - As you control your system, save the last N data point
 - Perform linear regression to estimate A_k and B_k
 - Use A_k and B_k to calculate new control law

- Self Tuning Regulator
 - Estimates A_k and B_k in real time
 - Recalculates control law (pole-place, MPC, PID) based on new A_k and B_k
- Gain Scheduling (Quasi-Adaptive)
 - Precomputes different laws (pole-place, MPC, PID)
 - Switches or interpolates between them based on estimated A_k and B_k





